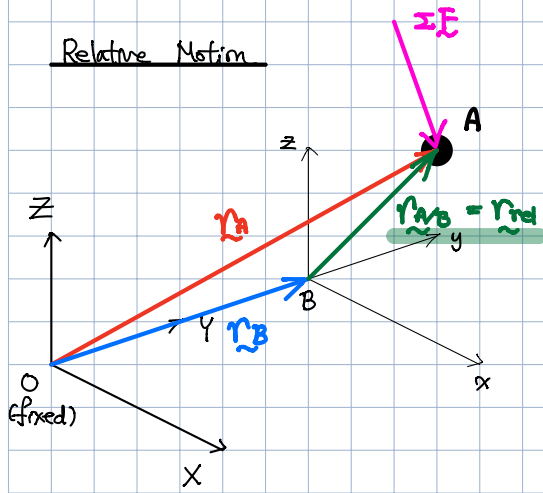


## Relative Motion



$$\underline{v}_A = \underline{v}_{AB} + \underline{v}_B$$

$$\underline{\ddot{r}}_A = \underline{\ddot{r}}_{AB} + \underline{\ddot{r}}_B$$

by Newton's 2nd Law:  $\Sigma \underline{F} = m \underline{\ddot{r}}_A$

$$= m (\underline{\ddot{r}}_{AB} + \underline{\ddot{r}}_B)$$

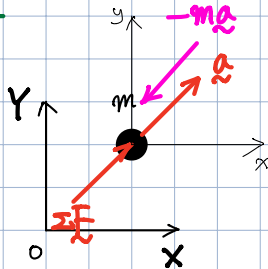
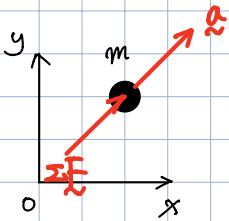
$$= m (\underline{\ddot{r}}_{rel} + \underline{\ddot{r}}_B)$$

if  $\underline{\ddot{r}}_B \neq 0 \rightarrow \Sigma \underline{F} \neq m \underline{\ddot{r}}_{rel}$

$\therefore \underline{\ddot{r}}_B = 0, \underline{\dot{r}}_B = \text{const} \rightarrow \Sigma \underline{F} = m \underline{\ddot{r}}_{rel} : \text{Newton's 2nd Law 적용가능!}$

$\hookrightarrow$  inertia system (= Newtonian frame of reference)

## \* D'Alembert Principle



$$\Sigma \underline{F} = m \underline{a}$$

$$\Sigma \underline{F} - m \underline{a} = 0$$

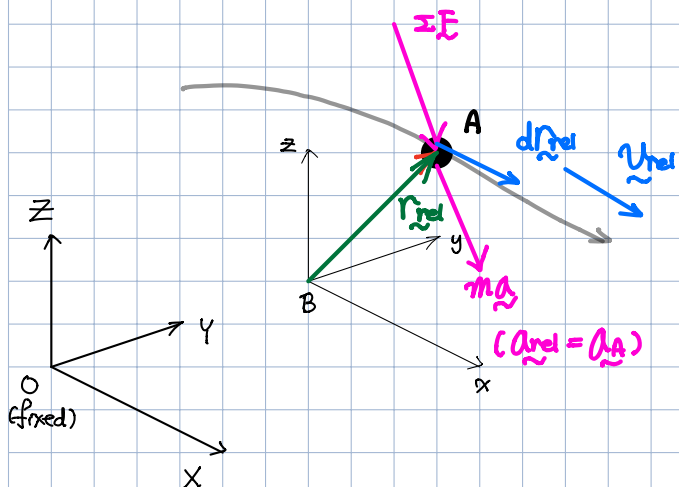
$\Sigma \underline{F}$  = actual force

$-m \underline{a}$  = inertia force

D'Alembert Principle !

1. 동적 평형을 의미
2. 동역학  $\rightarrow$  정역학.

## Constant-velocity, Non-rotating System (Translating System)



$$\Sigma \mathbf{F} = m (\mathbf{a}_B + \mathbf{a}_{rel})$$

constant velocity  
( $\mathbf{v}_B = \mathbf{0}$ )

$$dU_{rel} = \Sigma \mathbf{F} \cdot d\mathbf{r}_{rel}$$

$$= m \mathbf{a}_{rel} \cdot d\mathbf{r}_{rel}$$

$$= m U_{rel} \cdot dU_{rel}$$

$$= d\left(\frac{1}{2} m U_{rel}^2\right) \leftarrow T = \frac{1}{2} m u^2$$

$$\therefore dU_{rel} = dT_{rel}$$

$$\int_A^B dU_{rel} = \int_A^B dT_{rel}$$

$$U_{rel} = \Delta T_{rel}$$

the work-energy equation holds!

$$\Sigma \mathbf{F} = m \mathbf{a}_{rel}$$

$$\Sigma \mathbf{F} \cdot d\mathbf{t} = m \mathbf{a}_{rel} \cdot d\mathbf{t} \leftarrow \mathbf{a}_{rel} = \frac{d\mathbf{u}_{rel}}{dt}$$

$$= m d\mathbf{u}_{rel}$$

$$= d(m \mathbf{u}_{rel}) \quad (\because m = \text{const})$$

$$\int \Sigma \mathbf{F} dt = \int d(m \mathbf{u}_{rel}) = \int d\mathbf{G}_{rel}$$

$$\Sigma \mathbf{F} = \dot{\mathbf{G}}_{rel} \quad \int \Sigma \mathbf{F} dt = \Delta \mathbf{G}_{rel}$$

impulse-momentum equation holds

$$\Sigma \mathbf{M}_B = \mathbf{r}_{rel} \times \Sigma \mathbf{F} = \mathbf{r}_{rel} \times m \mathbf{a}_{rel}$$

$$\Sigma \mathbf{H}_B = \mathbf{r}_{rel} \times m \mathbf{u}_{rel}$$

$$\dot{\mathbf{H}}_B = \frac{d}{dt} (\mathbf{r}_{rel} \times m \mathbf{u}_{rel})$$

$$= \dot{\mathbf{r}}_{rel} \times m \mathbf{u}_{rel} + \mathbf{r}_{rel} \times m \dot{\mathbf{u}}_{rel} = \mathbf{r}_{rel} \times m \mathbf{a}_{rel} = \mathbf{r}_{rel} \times \Sigma \mathbf{F} = \Sigma \mathbf{M}_B$$

$$\therefore \Sigma \mathbf{M}_B = \dot{\mathbf{H}}_B$$

moment-angular momentum equation holds!

## constant velocity, translating systems

1. work-energy

$$U_{rel} = \Delta T_{rel} \quad 0$$

2. impulse-momentum

$$\int \Sigma \mathbf{F} dt = \Delta \mathbf{G}_{rel} \quad 0$$

3. moment-angular momentum

$$\Sigma \mathbf{M}_B = \dot{\mathbf{H}}_B \quad 0$$

4. work ( $dU = \Sigma \mathbf{F} \cdot d\mathbf{r}_A$ )  $\neq$  ( $dU_{rel} = \Sigma \mathbf{F} \cdot d\mathbf{r}_{rel}$ )

5. Kinetic Energy ( $T = \frac{1}{2} m u_A^2$ )  $\neq$  ( $T_{rel} = \frac{1}{2} m u_{rel}^2$ )

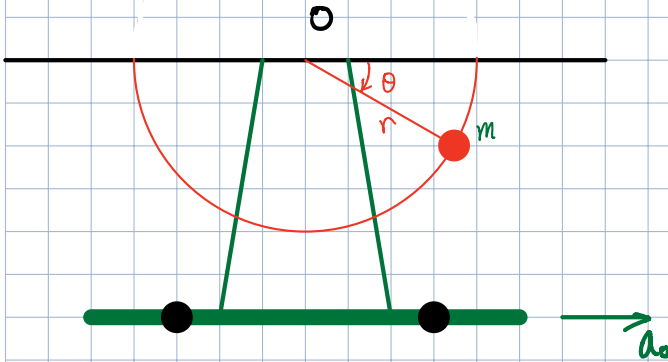
6. momentum ( $\mathbf{G} = m \mathbf{u}_A$ )  $\neq$  ( $\mathbf{G}_{rel} = m \mathbf{u}_{rel}$ )

### Example 3.32

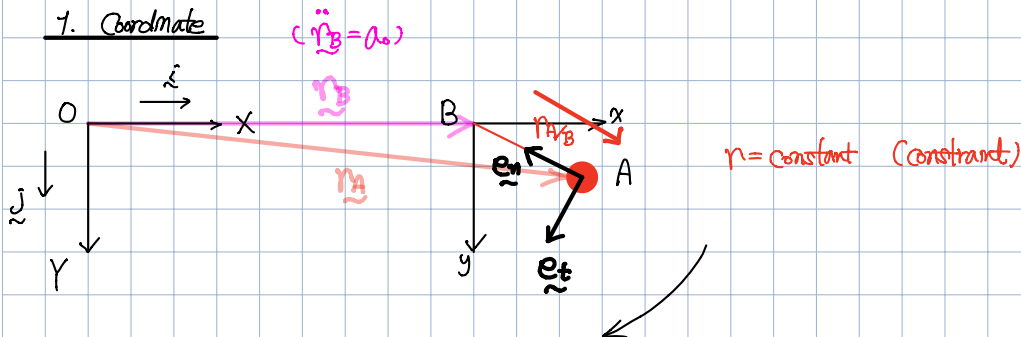
A pendulum of mass  $m$  and length  $r$  is mounted on the flat car, which has a constant horizontal  $a_0$ .

If the pendulum is released from rest to the flat car at the position  $\theta = 0$ ,

determine the expression for the tension  $T(\theta)$   
"  $T$  for  $\theta = \pi/2$ ,  $\theta = \pi$



#### 1. Coordinate



#### 2. Vectors

$$\underline{r}_A = \underline{r}_B + \underline{r}_{AB} \quad \leftarrow \quad \underline{r}_B = x\hat{i}, \quad \underline{r}_{AB} = r(-\hat{e}_n)$$

$$= x\hat{i} - r\hat{e}_n$$

$$\dot{\underline{r}}_A = \dot{x}\hat{i} - \dot{r}\hat{e}_n - r\dot{\hat{e}}_n$$

$\therefore r = \text{const} \Rightarrow \dot{r} = 0$

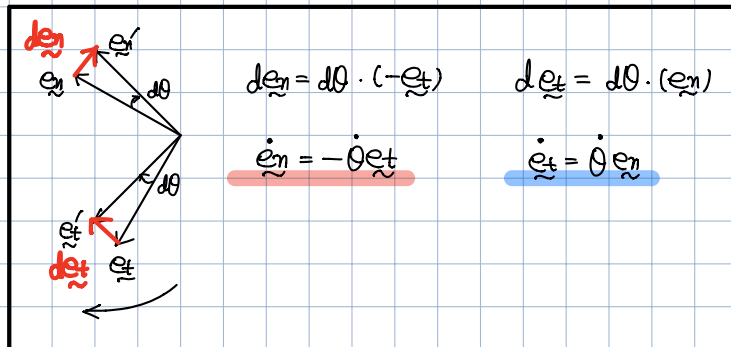
$$= \dot{x}\hat{i} + r\dot{\hat{e}}_t$$

$$\ddot{\underline{r}}_A = \ddot{x}\hat{i} + \dot{r}\dot{\hat{e}}_t + r\ddot{\hat{e}}_t + r\dot{\hat{e}}_t\dot{\hat{e}}_t$$

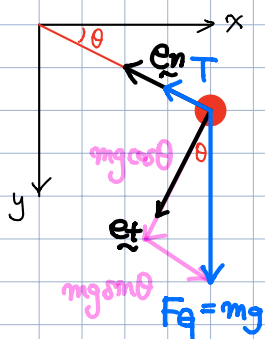
$= \dot{\hat{e}}_t$

$$= \ddot{x}\hat{i} + r\ddot{\hat{e}}_t + r\dot{\hat{e}}_t\dot{\hat{e}}_t$$

$= a_0$



#### 3. FBD



$$\underline{T} = T\hat{e}_n$$

$$\underline{F}_g = mg \sin \theta (-\hat{e}_n) + mg \cos \theta (\hat{e}_t)$$

#### 4. EOM

$$\Sigma \underline{F} = m\underline{a}_A$$

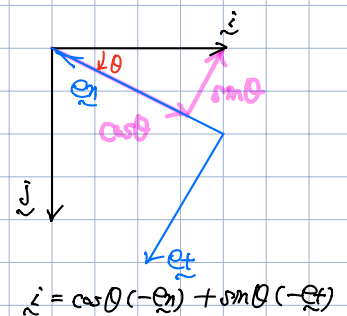
$$\Sigma \underline{F} = \underline{T} + \underline{F}_g = (T\hat{e}_n) + mg \sin \theta (-\hat{e}_n) + mg \cos \theta (\hat{e}_t)$$

$$m\underline{a}_A = m(a_0\hat{i} + r\ddot{\theta}\hat{e}_n + r\dot{\theta}^2\hat{e}_t)$$

$$= m(-a_0 \cos \theta \hat{e}_n - a_0 \sin \theta \hat{e}_t + r\ddot{\theta}\hat{e}_n + r\dot{\theta}^2\hat{e}_t)$$

$$T - mg \sin \theta = m(-a_0 \cos \theta + r\ddot{\theta}) \quad \dots \text{①}$$

$$mg \cos \theta = m(-a_0 \sin \theta + r\dot{\theta}^2) \quad \dots \text{②}$$



## 5. Calculation

$$\textcircled{1}: T = mg \sin \theta + m(-a_0 \cos \theta + r\ddot{\theta})$$

$$= mg \sin \theta - ma_0 \cos \theta + mr\ddot{\theta} \quad \leftarrow \textcircled{2}: \quad mg \cos \theta = m(-a_0 \sin \theta + r\ddot{\theta})$$

$$= mg \sin \theta - ma_0 \cos \theta$$

$$+ mr \left( \frac{2}{r} \cdot \{-a_0(\cos \theta - 1) + g \sin \theta\} \right)$$

$$= 2m(-a_0 \cos \theta + a_0) + 2mg \sin \theta$$

$$= -2ma_0 \cos \theta + 2ma_0 + 2mg \sin \theta$$

$$= -3ma_0 \cos \theta + 2ma_0 + 3mg \sin \theta$$

$$= m \{ 3g \sin \theta + a_0(2 - 3 \cos \theta) \}$$

~~~~~ Ans.

$$r\ddot{\theta} = a_0 \sin \theta + g \cos \theta \quad \leftarrow d\theta \stackrel{!}{=} \frac{dr}{r}$$

$$r\ddot{\theta} d\theta = a_0 \sin \theta d\theta + g \cos \theta d\theta \quad \leftarrow a dr = u du$$

$$r\dot{\theta} d\dot{\theta} = a_0 \sin \theta d\theta + g \cos \theta d\theta \quad \ddot{\theta} d\theta = \dot{\theta} d\dot{\theta}$$

$$\int_{\dot{\theta}(0)}^{\dot{\theta}(\theta)} r\dot{\theta} d\dot{\theta} = \int_{\theta(0)}^{\theta(\theta)} a_0 \sin \theta d\theta + \int_{\theta(0)}^{\theta(\theta)} g \cos \theta d\theta$$

$$\frac{1}{2} r \dot{\theta}^2 \Big|_0^\theta = \underline{a_0(-\cos \theta)} \Big|_0^\theta + g \sin \theta \Big|_0^\theta$$

$$\because \dot{\theta}(0) = 0 \quad \text{rest!} \quad = -a_0(\cos \theta - 1)$$

$$\frac{1}{2} r \dot{\theta}^2 = -a_0(\cos \theta - 1) + g \sin \theta$$

$$\dot{\theta}^2 = \frac{2}{r} \cdot \{-a_0(\cos \theta - 1) + g \sin \theta\}$$

$$\theta = \frac{\pi}{2} \rightarrow T(\theta = \frac{\pi}{2}) = m \{ 3g \sin \frac{\pi}{2} + a_0(2 - 3 \cos \frac{\pi}{2}) \}$$

$$= m(3g + 2a_0)$$

~~~~~ Ans.

$$\theta = \pi \rightarrow T(\theta = \pi) = m \{ 3g \sin \pi + a_0(2 - 3 \cos \pi) \}$$

$$= 5ma_0$$

~~~~~ Ans.